US009162791B2

(12) **United States Patent**
Gartrell, III

(10) **Patent No.: US 9,162,791 B2**
(45) **Date of Patent: Oct. 20, 2015**

(54) **SAFETY BRAKE DEVICE FOR THEATRE HOIST**

(71) Applicant: **The Rowland Company**, Philadelphia, PA (US)

(72) Inventor: **Robert Z. Gartrell, III**, Mt. Pleasant, SC (US)

(73) Assignee: **The Rowland Company**, Philadelphia, PA (US)

(*) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 88 days.

(21) Appl. No.: **13/845,696**

(22) Filed: **Mar. 18, 2013**

(65) **Prior Publication Data**

US 2013/0214225 A1 Aug. 22, 2013

Related U.S. Application Data

(62) Division of application No. 12/567,338, filed on Sep. 25, 2009, now Pat. No. 8,448,922.

(51) **Int. Cl.**

F16D 55/02	(2006.01)
B65D 5/00	(2006.01)
A63J 1/02	(2006.01)
B66D 5/14	(2006.01)

(52) **U.S. Cl.**

CPC . ***B65D 5/00*** (2013.01); ***A63J 1/028*** (2013.01); ***B66D 5/14*** (2013.01)

(58) **Field of Classification Search**

CPC B66D 5/00; B66D 5/14; A63J 1/028

USPC ..... 188/71.2, 71.1, 72.1, 73.1, 218 XL, 188/251 A; 192/12 R, 13 R, 16; 254/278, 254/318, 319, 321, 356, 375

See application file for complete search history.

(56) **References Cited**

U.S. PATENT DOCUMENTS

3,797,325	A	3/1974	Christison	
3,901,478	A	8/1975	Peterson	
4,175,727	A *	11/1979	Clarke	254/274
4,226,311	A *	10/1980	Johnson et al.	192/223
4,518,153	A *	5/1985	West et al.	254/274
4,615,418	A	10/1986	Atwell	
4,854,547	A	8/1989	Oliphant	
4,943,921	A	7/1990	Baltusis et al.	
5,141,085	A	8/1992	McCormick	
5,899,441	A	5/1999	Kuivamaki et al.	
6,630,416	B1 *	10/2003	Lam et al.	442/417
6,830,531	B1	12/2004	Koenig et al.	
6,889,958	B2	5/2005	Hoffend, Jr.	
8,448,922	B2 *	5/2013	Gartrell, III	254/321
8,613,428	B2 *	12/2013	Hoffend, III	254/374
2004/0168865	A1	9/2004	Kuivamaki et al.	

FOREIGN PATENT DOCUMENTS

JP	09229113	A *	9/1997
KR	101180271	B1 *	9/2012

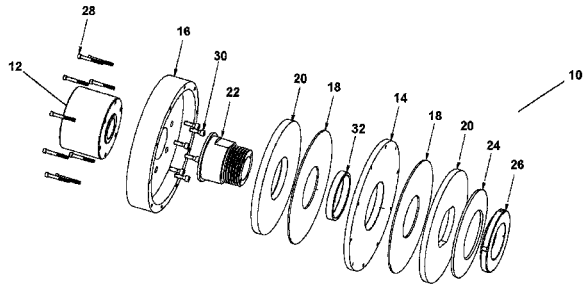
* cited by examiner

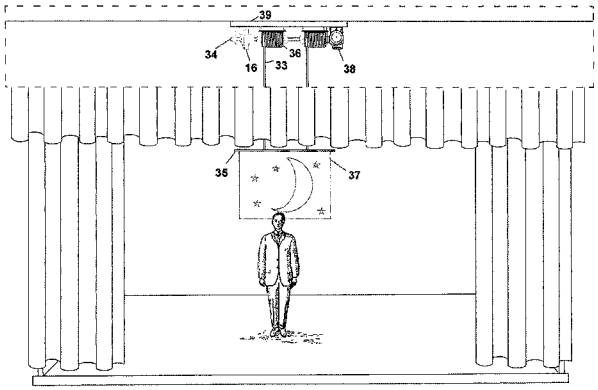
Primary Examiner — Pamela Rodriguez
(74) *Attorney, Agent, or Firm* — Paul & Paul

(57) **ABSTRACT**

A safety brake device for a theatre hoist to prevent the uncontrolled release of a load that is suspended above or below people includes an overrunning clutch and a torque disc. The torque disc only rotates with the overrunning clutch when the load is lowered, but must overcome friction forces applied to the surface of the torque disc to do so. The friction forces are constantly applied to the torque disc by maintaining friction material in contact with the torque disc. The friction material is a non-asbestos, non-metallic composite saturated with a lubricant.

7 Claims, 7 Drawing Sheets





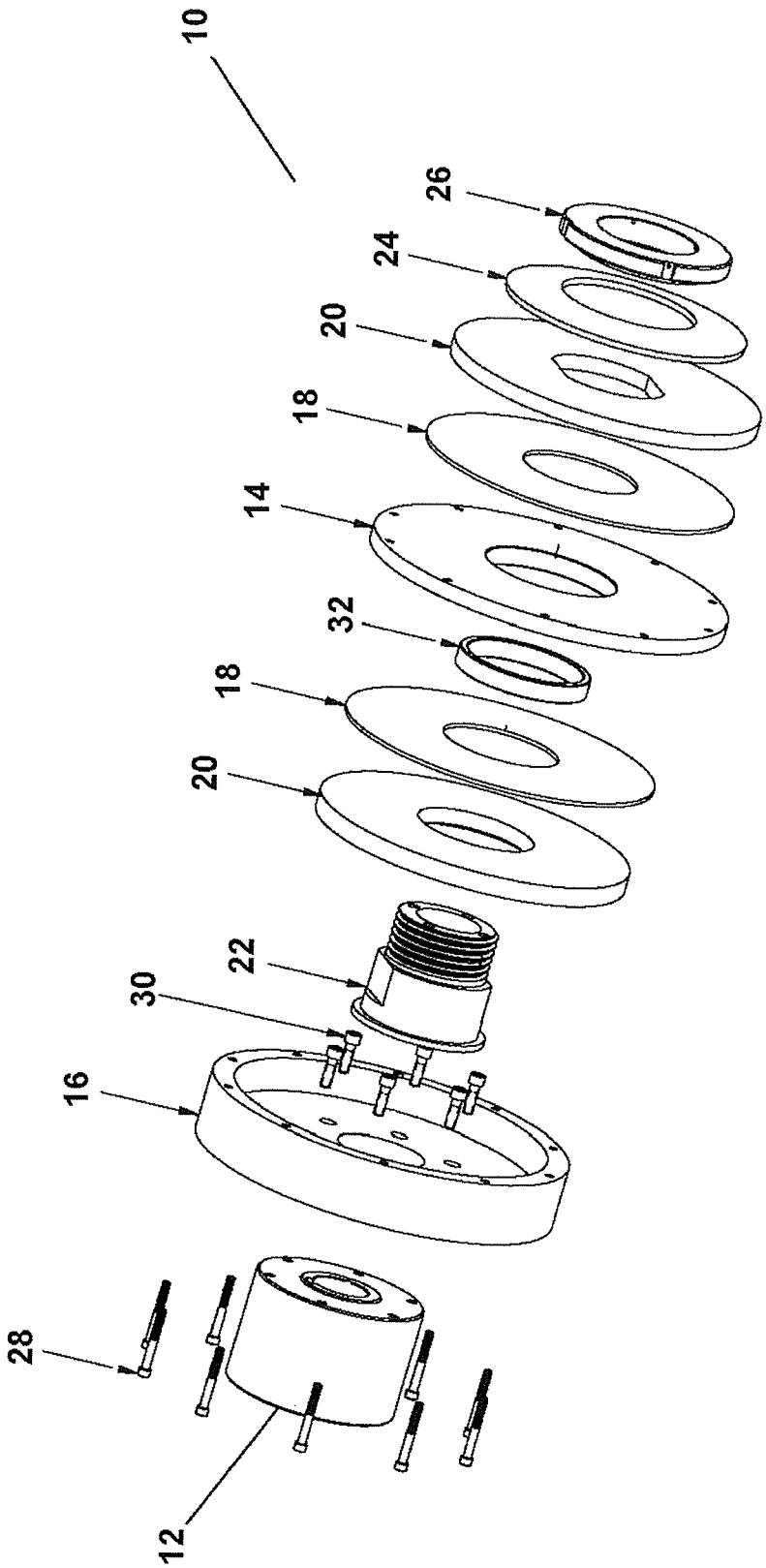


Fig. 1

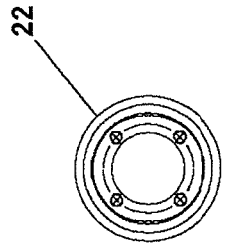


Fig. 4

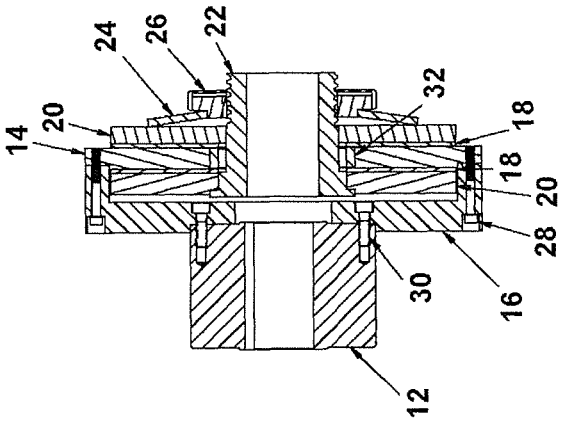


Fig. 3

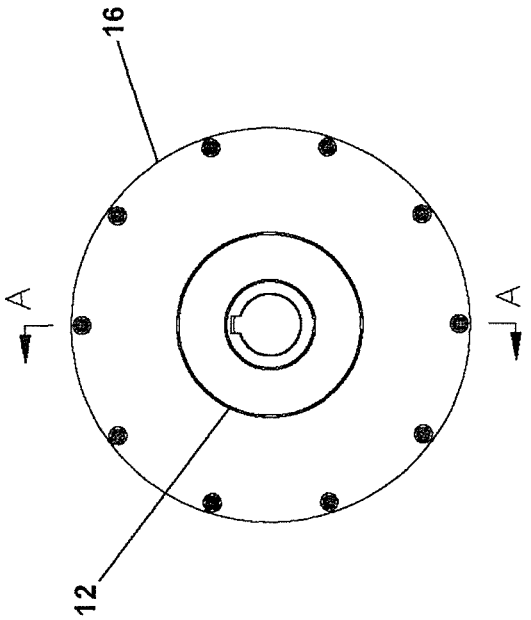


Fig. 2

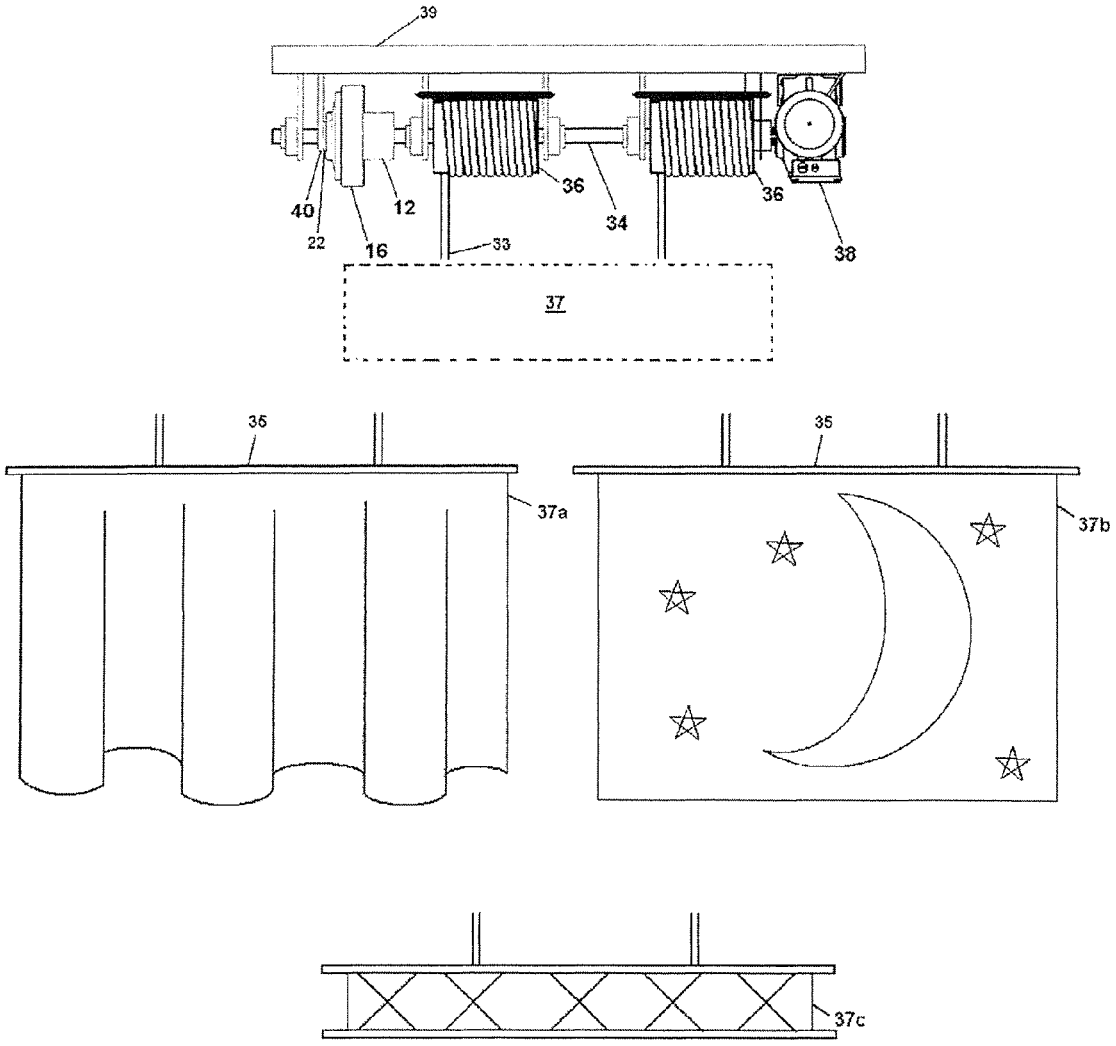


FIG. 5a

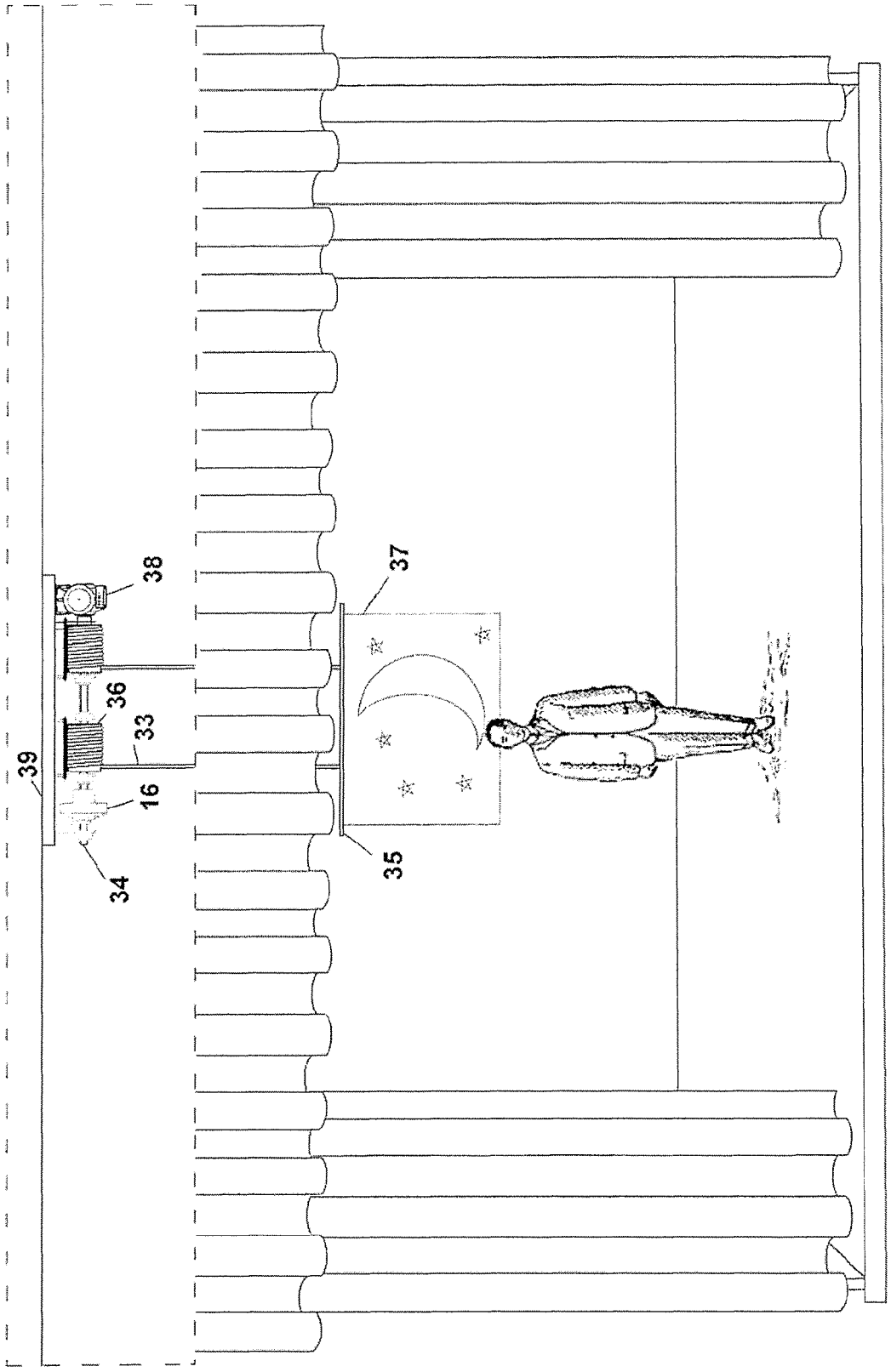


FIG. 5b

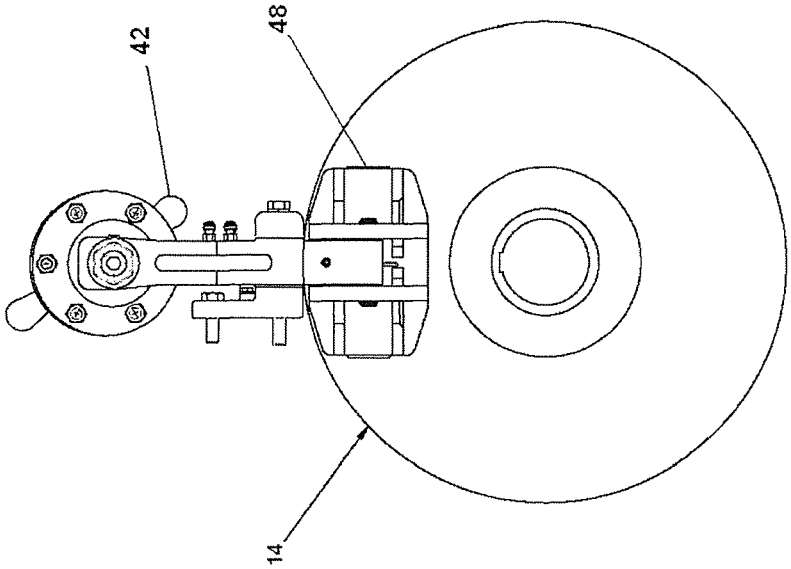


Fig. 7

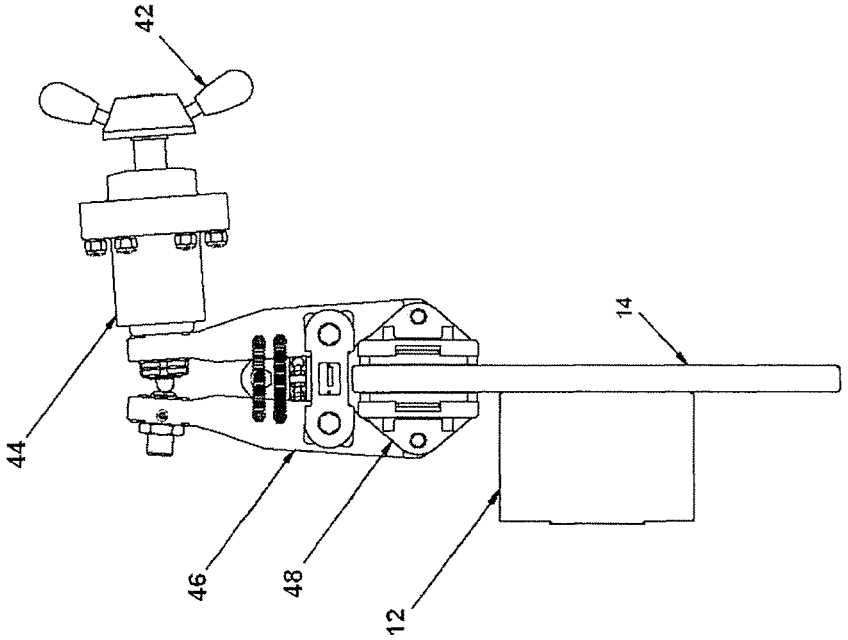


Fig. 6

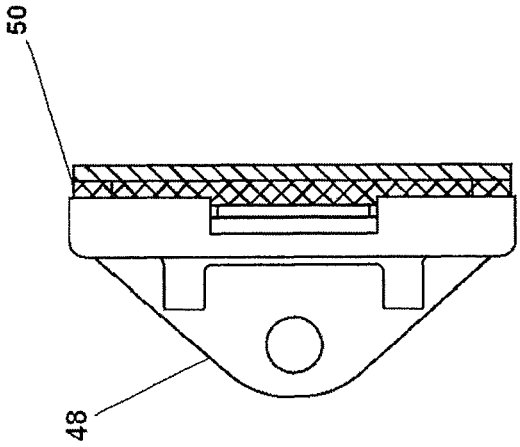


Fig. 9

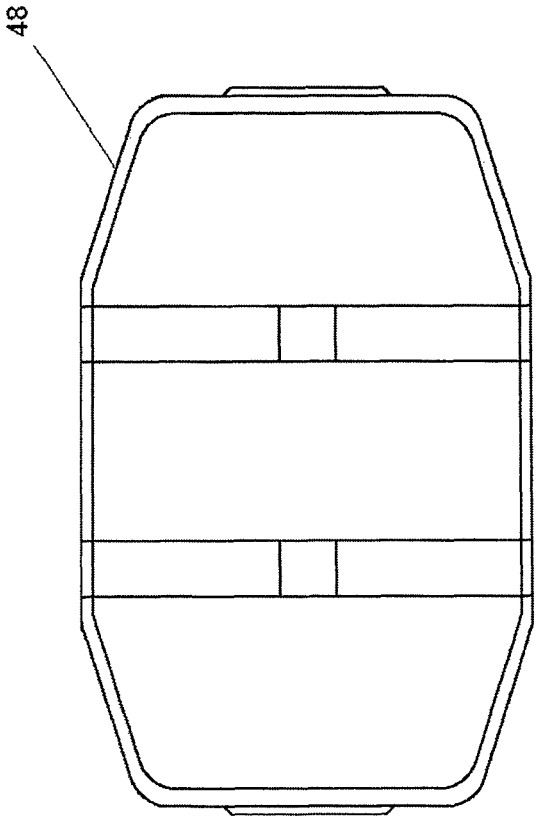


Fig. 8

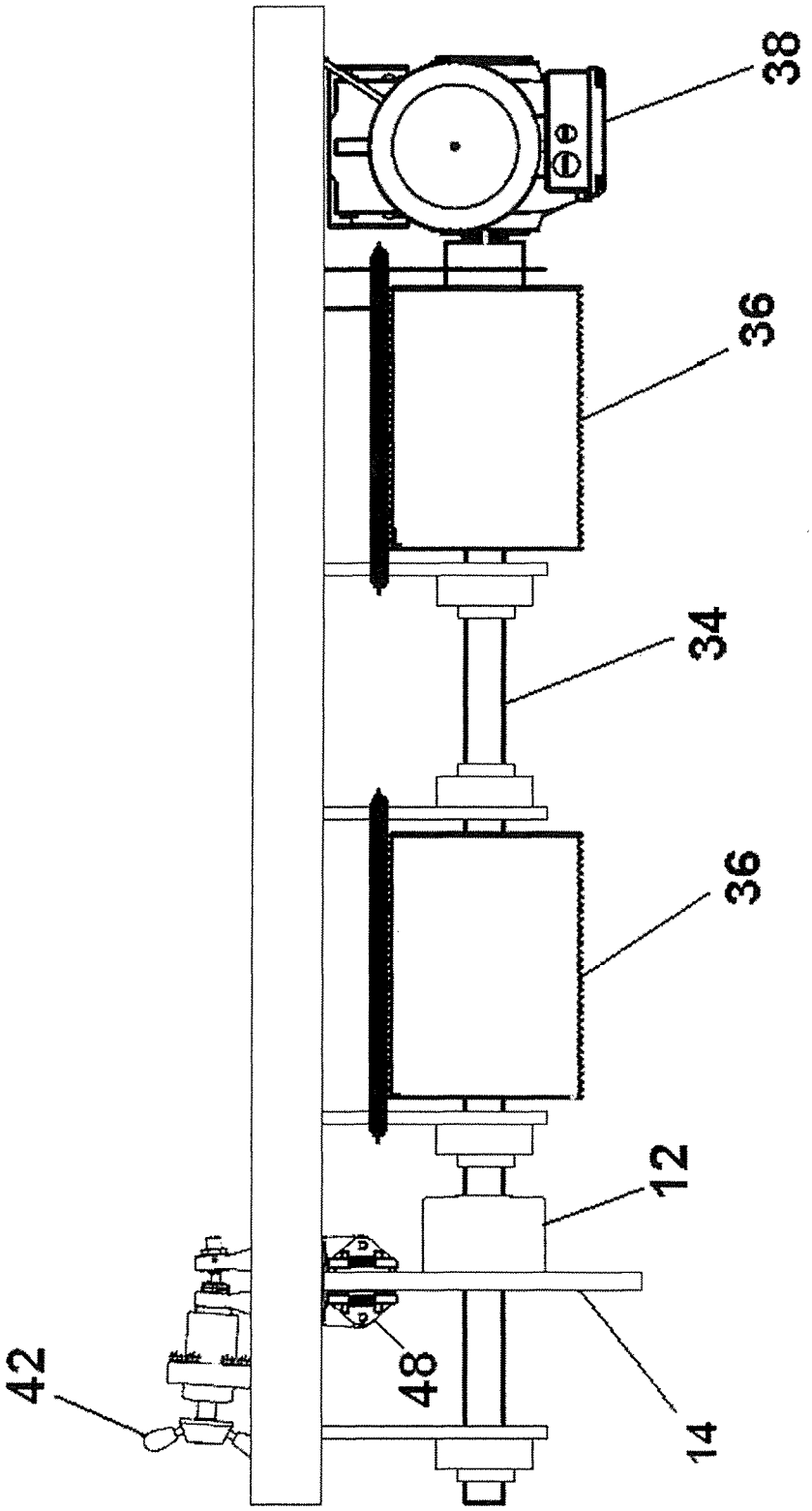


Fig. 10

SAFETY BRAKE DEVICE FOR THEATRE HOIST

CROSS-REFERENCE TO RELATED APPLICATIONS

This application is a division of U.S. application Ser. No. 12/567,338, filed Sep. 25, 2009, now U.S. Pat. No. 8,448,922 B2, issued on May 28, 2013.

BACKGROUND OF THE INVENTION

1. Field of Invention

The present invention relates to safety brakes and, more specifically, to a safety brake device applied to theatre hoists that lift and maintain heavy loads suspended.

2. Description of Prior Art

Hoists that lift loads in a vertical direction are used in many industries for a variety of applications. For theatrical settings, athletic and entertainment arenas, overhead lifting with higher safety standards are routinely required because hoists are lifting loads directly over human beings. It is also common for portions of the staging in these theatrical settings to be lifted. Similar safety standards are required in these instances because people may be standing on the portion of the stage being lifted.

Live performances in a theater typically employ a number of curtains and backdrops to convey to the audience different settings, environments, moods, and the like. These curtains and backdrops must be changed throughout the course of a performance within a fairly short time frame without interrupting the performance. Typically this is done by raising a particular backdrop above the stage and out of sight of the audience when it is not being used. When a particular backdrop is needed, it is lowered into place on the stage.

Theatrical backdrops and curtains are typically suspended from battens, which are pipes or trusses that span the width of the stage. Battens can be 20 feet or more in length, depending on the size of the stage. As should be apparent, the weight of the battens and the items suspended from them can have substantial weight. As the weight of the load increases so does the power required to raise the load. Counterweights are employed to balance the load of the batten and its associated load. However, if the load is not closely balanced or if there is a failure in the motorized drive lifting the hoist, the system may get out of control, dropping the load or the counterweight, causing injury or death to people nearby and/or collateral damage.

Therefore, because of the risk of hoist failure, there is a need for a safety device to prevent the uncontrolled release of heavy loads and staging that are either supported above or below human beings.

SUMMARY

The present invention comprises a combination overrunning clutch, torque disc, and friction material for preventing the uncontrolled lowering of a load. An axle connected to a motorized drive engages the overrunning running clutch. A torque disc fixedly attached to the overrunning clutch will rotate with the overrunning clutch when a load is lowered; however, resistance against rotation is generated by a set of fixed friction discs applying pressure to the sides of the torque disc. In order to lower a load, the motorized drive must overcome the friction forces applied to the side surfaces of the torque disc, thus enabling the axle to rotate.

None of the prior art devices are seen to offer the advantages of the present invention that will become apparent from the detailed description of the invention provided below.

It is an advantage of the present invention to provide a safety brake device that prevents the uncontrolled release of a suspended load.

It is a further advantage of the present invention to provide a safety brake device that provides smooth consistent resistance without producing excessive noise during operation.

It is a further advantage of the present invention to provide a safety brake device that uses a friction material that operates consistently at different temperatures and irregular use.

It is a further advantage of the present invention to provide a safety brake device that uses a friction material that is long wearing, thereby reducing the need for adjustment.

It is an advantage of the present invention to provide device of simple design and manufacture that can be fitted to current hoisting systems.

Other objects and advantages of the present invention will in part be obvious, and in part appear hereinafter.

BRIEF DESCRIPTION OF THE DRAWINGS

The present invention will be more fully understood and appreciated by reading the following Detailed Description in conjunction with the accompanying drawings, in which:

FIG. **1** is an exploded perspective view of a first embodiment of the present invention.

FIG. **2** is left-side elevational view of the first embodiment of FIG. **1**.

FIG. **3** is a cross-sectional view of the first embodiment along line "A-A" of FIG. **2**.

FIG. **4** is a right-side elevational view of a hub for the first embodiment of the present invention.

FIG. **5***a* is an elevational view of a first embodiment of the present invention mounted on an axle and suspended from a frame.

FIG. **5***b* is an elevational view of an installed first embodiment of the present invention above a stage.

FIG. **6** is a front elevational view of a second embodiment of the present invention.

FIG. **7** is a left-side elevational view of the second embodiment of FIG. **6**.

FIG. **8** is a rear elevational view of a friction disc of the second embodiment of the present invention.

FIG. **9** is a left-side elevational view of the friction disc of FIG. **8**.

FIG. **10** is an elevational view of an installed second embodiment of the present invention.

DETAILED DESCRIPTION

A first embodiment of the present invention is shown in FIGS. **1** through **4**. The first embodiment of the safety brake device comprises an overrunning clutch **12** and a disc assembly **10**. The overrunning clutch **12** may be of any suitable design known in the art, such as a ramp and roller or sprag, and includes a keyed bore enabling the inner race of the overrunning clutch to rotate with an axle installed through the keyed bore. The clutch is designed and installed such that the outer race will rotate with the inner race and axle only when a load is lowered.

The disc assembly **10** comprises a hub **22** which is installed adjacent to the overrunning clutch **12** along the axle. The hub **22** also has a bore enabling it to be installed onto the axle; however, the diameter of the bore is not keyed and is larger than the diameter of the axle, so that the hub **22**, if fixed, will

not rotate with the axle. Mounted onto the hub is a torque disc **14** sandwiched between a set of friction discs **18** and backing plates **20**. A securing means is required to constantly maintain a force that presses the friction discs **18** against the sides of the torque disc **14**. In the first embodiment, the securing means comprises a nut **26** and Belleville washer **24** that is screwed onto a threaded end of the hub **22**, such that the Belleville washer is pressed against one face of a backing plate **20**. The disc assembly **10** further comprises an adaptor **16** fixed to the torque disc **14**. A first set of screws **28** are used to attach the torque disc **14** to the adaptor **16**.

A second set of screws **30** are used to attach the adaptor **16** to the outer race of the overrunning clutch **12**. The adaptor **16** and torque disc **14** are fixedly attached to the outer race of the overrunning clutch **12**, so that the three elements will rotate together when lowering a load; however, the friction discs **18** and backing plates **20** remain fixed on the hub **22** and will not rotate with the torque disc **14**, thus generating a friction forces between the torque disc **14** and friction discs **18** when a load is lowered. The disc assembly optionally includes a bearing **32** that keeps the torque disc **14** aligned with the friction discs **18** and prevent uneven wear of the friction discs.

Referring now to FIGS. **5***a* and **5***b*, a typical environment in which the present invention may be installed is shown. The safety brake device is mounted on the opposite end of an axle **34** from a motorized drive **38** and suspended from a frame **39** above a stage. The dashed outline in FIG. **5***b* provides a cut-away view of the area above the stage where the first embodiment of the invention is typically installed. The exposed face of the hub **22** is attached to a bracket **40** which keeps the hub **22** fixed as the axle **34** rotates. A set of winch drums **36** fixed onto the axle **34** may also be present. Cables **33** wound around the winch drums **36** are attached to a batten **35** from which a load **37**, such as a curtain **37***a* or theatrical scenery **37***b*, is suspended. The cables **33** can also be directly attached to a load, such as a platform **37***c*. When the motorized drive **38** rotates the axle **34** to lift a load, the inner race of the overrunning clutch **12** rotates with the axle **34**, but the remaining parts of the safety brake device remain fixed.

Once a load is suspended, the motorized drive **38** stops. The weight of the load will force the axle **34** to rotate in the opposite direction to lower the load; however, at this instance, the overrunning clutch **12** will lock, so that the outer race, adaptor **16**, and torque disc **14** will attempt to rotate, but will be held in place because of the friction forces between the torque disc **14** and the friction discs **18** which remain stationary with the hub **22**. If a friction disc is selected such that the friction forces are equal to the gravitational forces of the load, the motorized drive is not taxed and only a slight application of rotational force to the axle is necessary to set the load in motion.

An appropriate friction material must be selected for the friction discs **18** which has a low differential between static and dynamic coefficients of friction, such that a motorized drive is not heavily taxed when started and loads may be raised and lowered at a slow speed. It is preferred that the ratio between the static coefficient of friction and the dynamic coefficient of friction for the friction material be equal to or greater than 1.05 and less than or equal to 1.15. The friction material needs to provide smooth consistent resistance without producing any squeal, as excessive noise would be unwanted during a performance. Eliminating squeal can be achieved by saturating the friction material with a lubricant. Given the often unpredictable system usage, the friction material needs to be consistent at different temperatures and irregular use. Finally, the material needs to be long wearing reducing the need for adjustment and replacement. Any frictional material known in the art to include these characteristics, for example the frictional materials disclosed in U.S. Pat. No. 6,630,416, the disclosure of which is incorporated herein by reference, is acceptable.

Referring now to FIGS. **6** though **9**, a second embodiment of the invention is disclosed wherein the disc assembly has been replaced with a caliper and pad assembly. The second embodiment of the invention does not require the use of an adaptor as the torque disc **14** is secured directly to the overrunning clutch **12**. The friction material is now in the form of a pair of friction pads that sandwich the torque disc **14**. The friction pads are comprised of a shoe **48** to which the friction material **50** is bonded. An intermediate backing layer may be employed between the shoe **48** and the friction material **50**. The friction pad shoes **48** are attached to a caliper **44** which applies the necessary force to the sides of the torque disc **14**. Turning the knob **42** of the caliper **44** increases the distance between the ends of the caliper arms **46**. Because the caliper arms **46** are pivotally connected, the distance is decreased between the opposite ends of the caliper arms **46** to which the friction pads are attached. The knob **42** is turned and left in position to constantly maintain a force on the sides of the torque disc **14**. FIG. **10** demonstrates second embodiment of the invention installed in the same typical environment shown in FIG. **5**. The caliper **44** is braced to the frame to which the motorized drive and axle are suspended. The torque disc **14** has a bore enabling it to be installed onto the axle; however, the diameter of the bore is not keyed and is larger than the diameter of the axle, so that the torque disc **14** will only rotate with the outer race of the overrunning clutch **12** when a load is lowered. The friction forces applied by the friction pads on the caliper **44** should be equal to the gravitational forces of the load, such that the motorized drive is not taxed and only a slight application of rotational force to the axle is necessary to set the load in motion.

Thus, there has been described and illustrated herein a safety brake device that prevents the uncontrolled release of a suspended load. However, those skilled in the art will recognize that many modifications and variations besides those mentioned specifically may be made in the technique described herein without departing substantially from the spirit and scope of the present invention. For example, the safety brake device may be designed as a drum brake wherein the friction material is in the shape of a collar that applies frictional forces to the circumference of the torque disc. Accordingly, it should be clearly understood that the forms of the invention described herein are exemplary only, and are not intended as a limitation on the scope of the present invention.

What is claimed is:

1. A safety brake device capable of being mounted on to a shaft of a theatre hoist to prevent the uncontrolled release of a suspended load, said safety brake device comprising:

an overrunning clutch having an inner race, an outer race, and said inner race having a first bore along a center-axis of said overrunning clutch to receive the shaft, said first bore of said inner race being a keyed bore that is dimensioned and configured for keyed attachment to said shaft such that said inner race rotates with the shaft when said inner race is keyed to the shaft;

a torque disc fixedly attached to said outer race of said overrunning clutch, said torque disc having a first surface perpendicular to said center-axis, a second surface parallel to said first surface, and a second bore aligned with said center-axis,

wherein the shaft has a diameter, wherein said second bore of said torque disc has a diameter greater than the diameter of the shaft to allow the shaft to extend through said

second bore of said torque disc while said torque disc can rotate relative to the shaft when said inner race is keyed to the shaft, and wherein the shaft extends through said second bore of said torque disc when said inner race is keyed to the shaft; and

a caliper assembly, wherein said caliper assembly comprises a first and second friction pad, wherein said caliper presses said first friction pad against said first surface of said torque disc and said second friction pad against said second surface of said torque disc.

2. The safety brake device of claim **1**, wherein said caliper assembly further comprises a first and second caliper arm and said first and second friction pads are bonded to a first and second shoe, respectively, wherein said first shoe is attached to an end of said first caliper arm and said second shoe is attached to an end of said second caliper arm.

3. The safety brake device of claim **1**, wherein said overrunning clutch is one of a ramp and roller overrunning clutch or a sprag overrunning clutch.

4. The safety brake device of claim **1**, wherein the first and second friction pads are made of a material that is capable of generating static friction greater than or equal to the weight of the load suspended by the hoist employing the safety brake device.

5. The safety brake device of claim **1**, wherein the first and second friction pads are made of a material that is at least partially saturated with a lubricant.

6. The safety brake device of claim **1** wherein the first and second friction pads are made of a material having a ratio of static coefficient of friction to dynamic coefficient of friction from 1.05 to 1.15.

7. The safety brake device of claim **1** in combination with the shaft attached to a motorized drive and at least one winch drum, wherein said safety brake device and said at least one winch drum is mounted on said shaft and said at least one winch drum carries a cable secured to the load.

* * * * *

UNITED STATES PATENT AND TRADEMARK OFFICE

CERTIFICATE OF CORRECTION

PATENT NO. : 9,162,791 B2
APPLICATION NO. : 13/845696
DATED : October 20, 2015
INVENTOR(S) : Robert Z. Gartrell, III

It is certified that error appears in the above-identified patent and that said Letters Patent is hereby corrected as shown below:

Title Page:
Item 72 the address of the Inventor, change "Mt. Pleasant" to -- Charleston --

Signed and Sealed this
Twenty-first Day of June, 2016

Michelle K. Lee

Michelle K. Lee
Director of the United States Patent and Trademark Office